

Operation of CHP Systems at UMD, Measurement, Data Evaluation and Technology Transfer

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December 2-4, 2003

Washington, DC



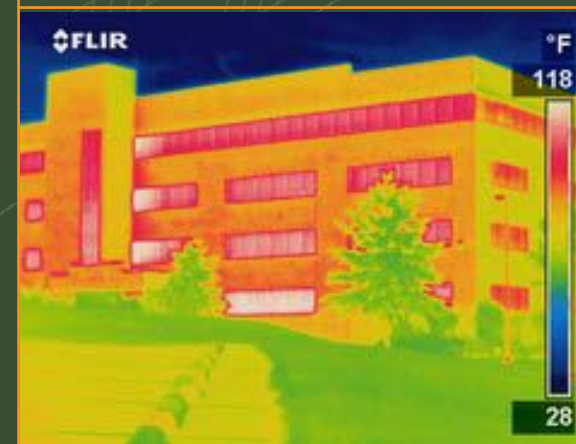
- Introduction
- Objectives
- Project Team/Partnerships
- Operation
- CHP Know-How/Accomplishments
- Future work
- Summary



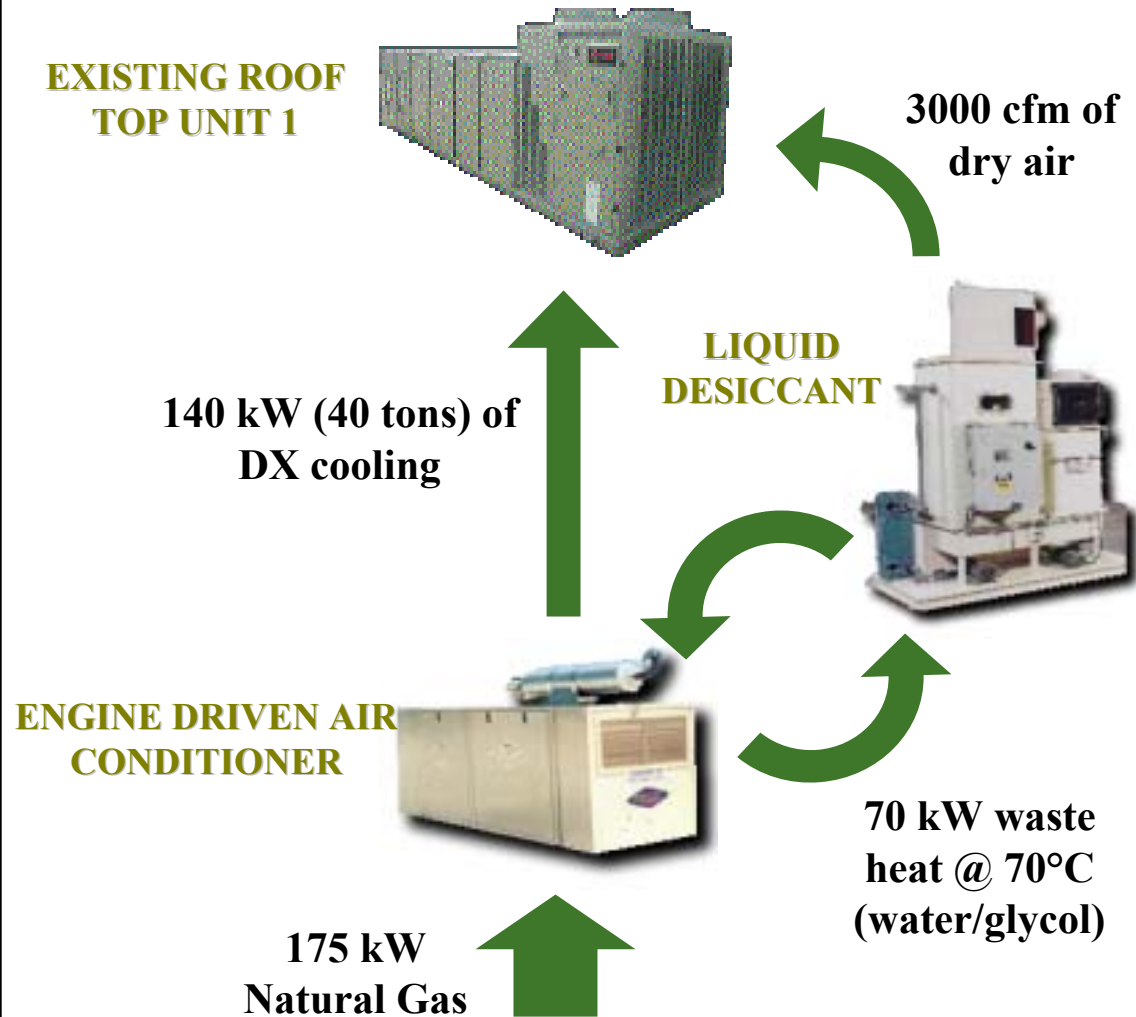
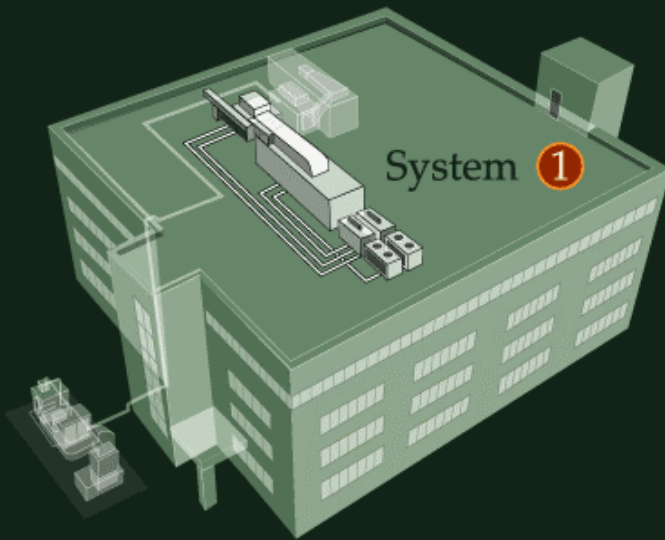
- Medium sized office building on UMD campus, 51,000 ft²
- Represents 23% of U.S. office space
- CHP systems installed late 2000
- 2 completely separate air conditioned zones within building
- CHP systems only operate in the cooling season as heating is distributed electric reheat
- Both systems have sensible and latent cooling components



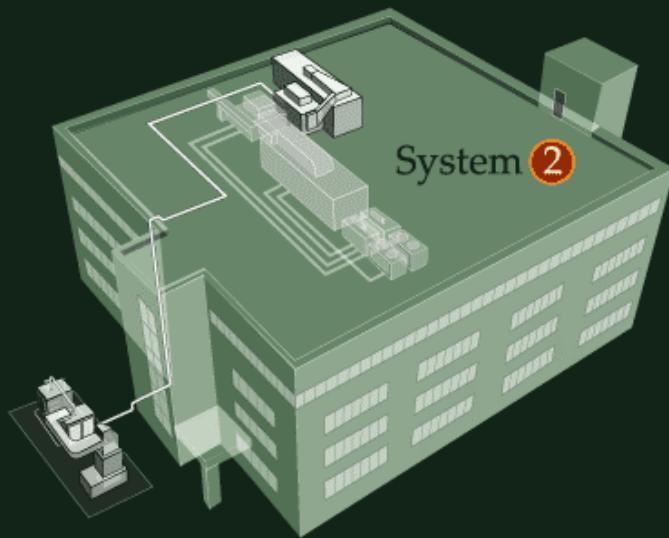
The Chesapeake Building at UMD



Introduction - CHP System 1



Introduction - CHP System 2



**SOLID
DESICCANT**



**EXISTING ROOF
TOP UNIT**

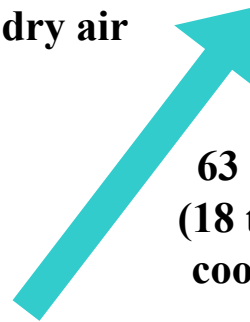


3000 cfm
dry air

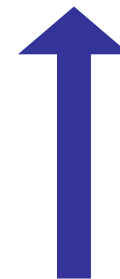
60 kW
Waste Heat
@105°C



63 kW
(18 tons)
cooling



60 kW
Power



150 kW
Waste Heat
@300°C



**ABSORPTION
CHILLER**



220 kW
Natural Gas



MICROTURBINE

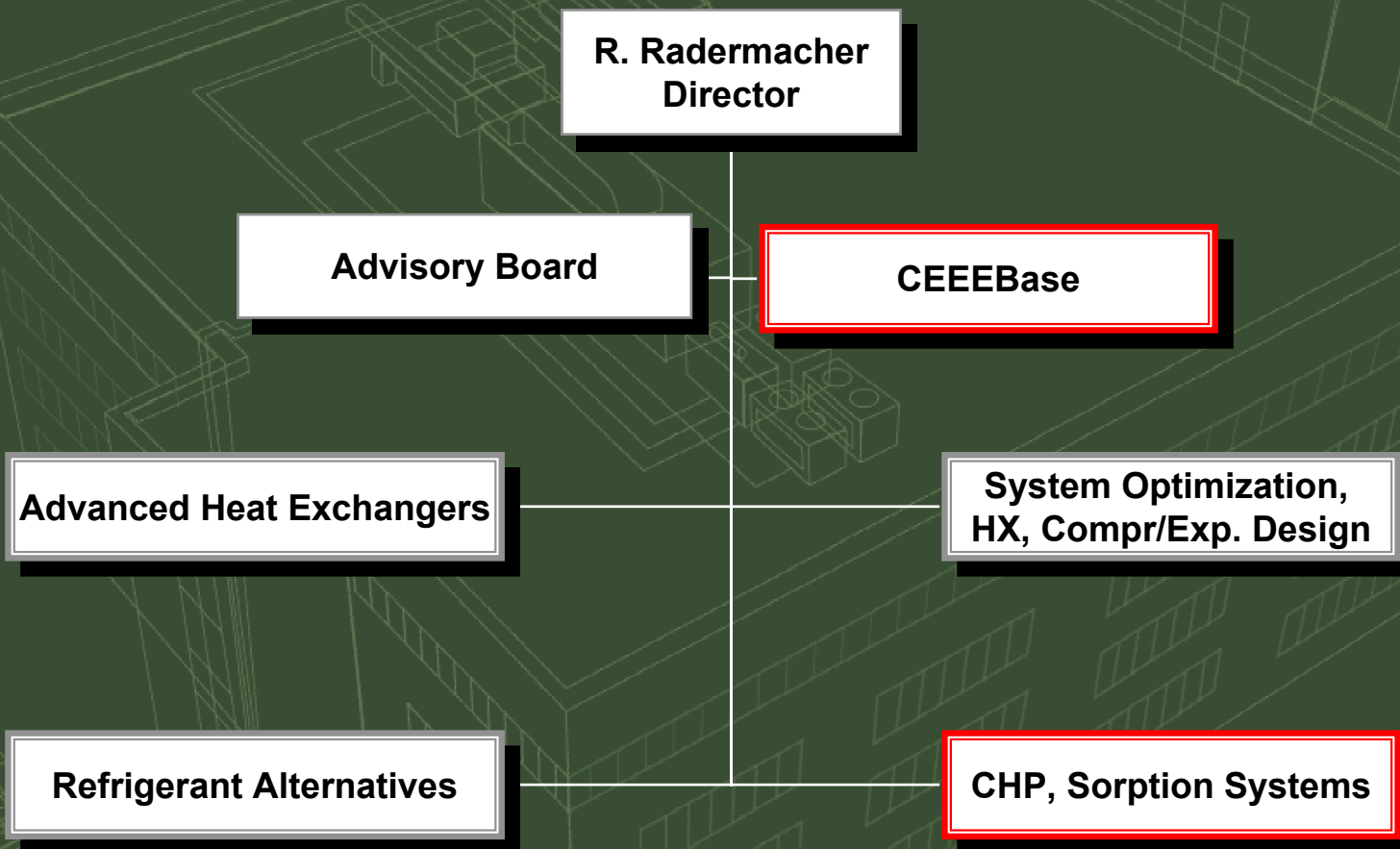
- Benchmark CHP equipment performance
- Benchmark integrated equipment system performance in an occupied building
- Develop and verify computer models
- Identify component and system improvements for current packaged CHP manufacturers, “next generation” products and fuels
- Technology Transfer



CHP System 2 at the Chesapeake Building

CEEE Organization

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CEEE Consortium Partners

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1	Baltimore Aircoil	6	DOE/ORNL	12	Thermoflow
2	Broad	7	Heatcraft	13	Trane
3	Capstone	8	Honeywell	14	Tridium
4	Daikin	9	Kathabar	15	Trigen
5	DTE Energy	10	PEPCO	16	Trion
		11	Propane Res. Council	17	Sanyo

Trigen Contribution and Fellowships



U.S. Department of Energy
Energy Efficiency
and Renewable Energy

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www.enme.umd.edu/ceee/bchp

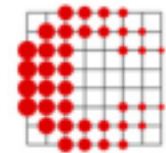


UNIVERSITY OF
MARYLAND



INTEGRATED EQUIPMENT TEST CENTER

9



CAPRON



TRiD!UM™

Honeywell



Pacific Northwest
National Laboratory



PROPANE
EXCEPTIONAL ENERGY™

KATHABAR®

SANYO

TRION®



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UNIVERSITY OF
MARYLAND



- 180 sensors throughout building
- Temperature, humidity and flow rate sensors calibrated
- Data collected and statistically sampled for calibration in HP VEE software
- DAS automatically records all test points once/minute.



DAS sensor board collects data continuously to monitor building performance

Operation - Weekly Test Schedule

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Weekday	Test Item	Changeable Variables	Fixed Variables	Purpose
Monday	Baseline: run MT at 10-60kWe power output and run chiller and ATS together	MT power (10kW-60 kW)	Chiller at default settings ATS burner is off	Set CHP baseline
Tuesday	ATS Integration test: ATS runs on the exhaust and burner	Burner temperature	MT=60kWe, Chiller at default settings	Make up heat for ATS
Wednesday	Air cooled simulation tests	Condenser return temp up to 120°F	MT=60kWe, Chiller at default settings	Air cooled similarity tests
Thursday	Broad Integration test: with ATS off	Cooling water temp. (~105°F) Chilled water temp. (~45°F) Exhaust temp. (~550°F)	MT=60kWe	Investigate back pressure effect
	ATS on burner alone	ATS on/off		ATS baseline and transient test to investigate the impact on RTU2
Friday	Broad Integration test: with ATS on	Cooling water temp. (~105°F) Chilled water temp. (~45°F) Exhaust temp. (~550°F) Cooling load (adjust the valve)	MT=60kWe, ATS on exhaust	In CHP mode, investigate the effect of several variables



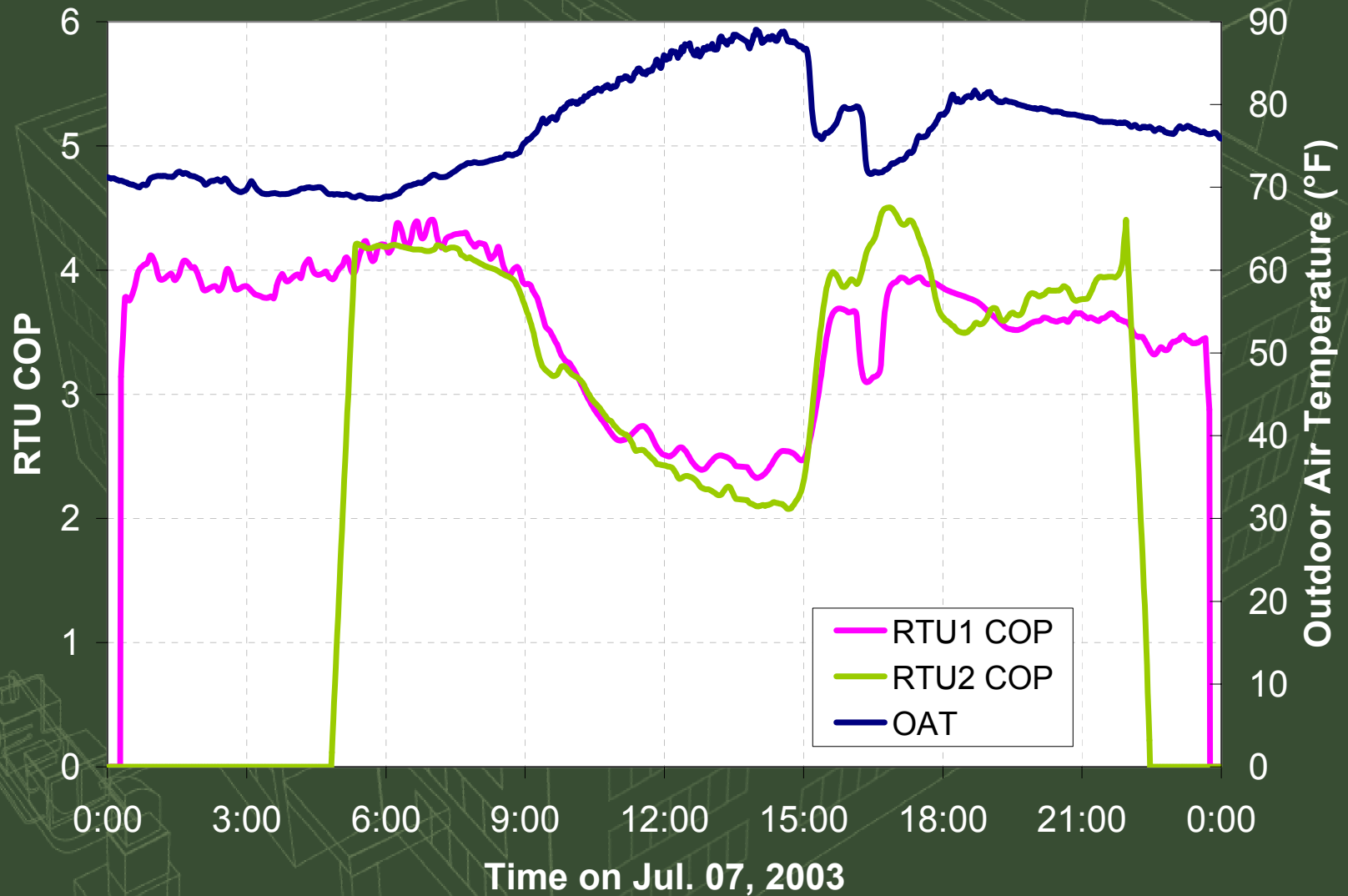
- Set weekly test schedule
 - To make use of different weather conditions and compensate the deficiency of real building tests.
 - To compare different CHP operation in 2 similar days.
- Recorded comprehensive test log.
- Ran CHP System 2 on natural gas for 450 hrs (equal to 35 hrs/week) in Jun. ~ Aug. 2003
- Ran microturbine on propane since fall, 8 hrs/day
- Enthalpy wheels comparison test



Propane tank outside the Chesapeake building

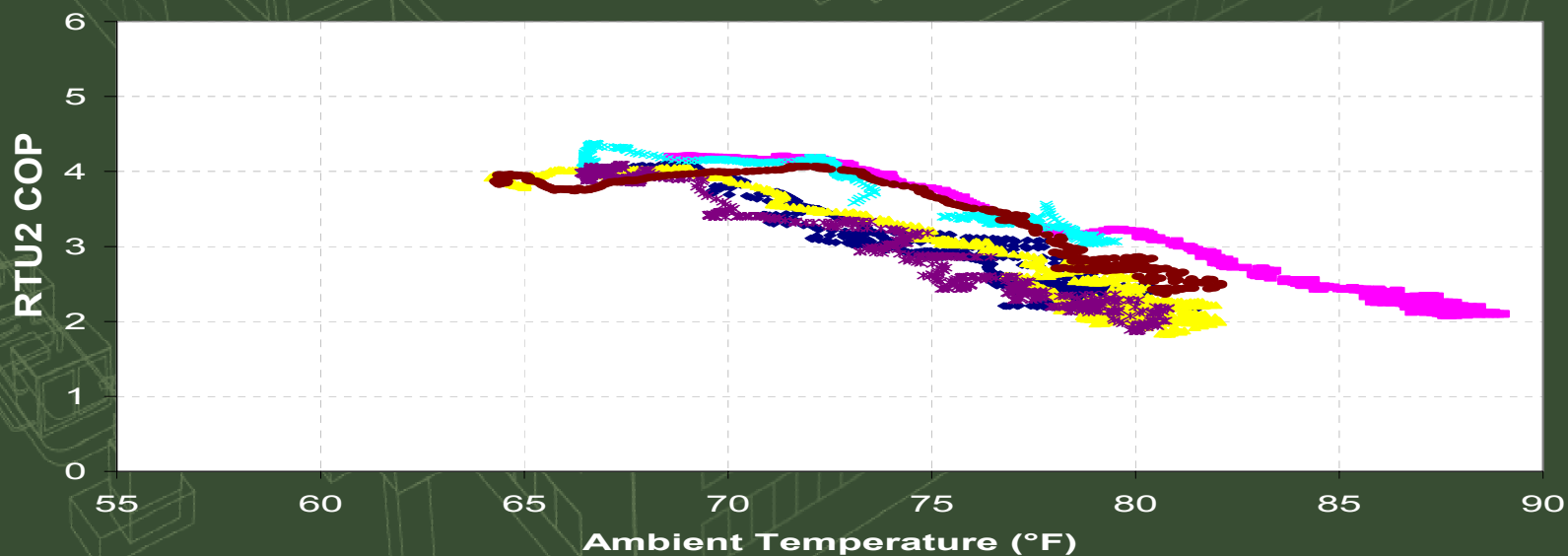
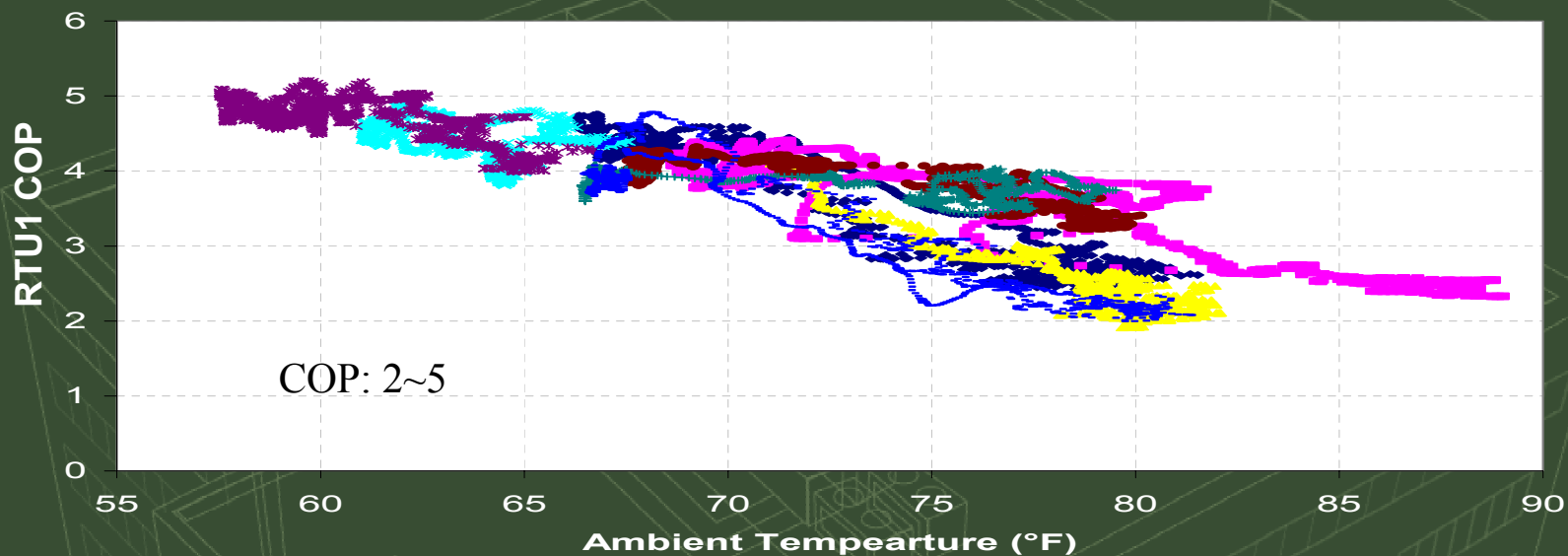
Roof Top Unit – Electric Vapor Compression

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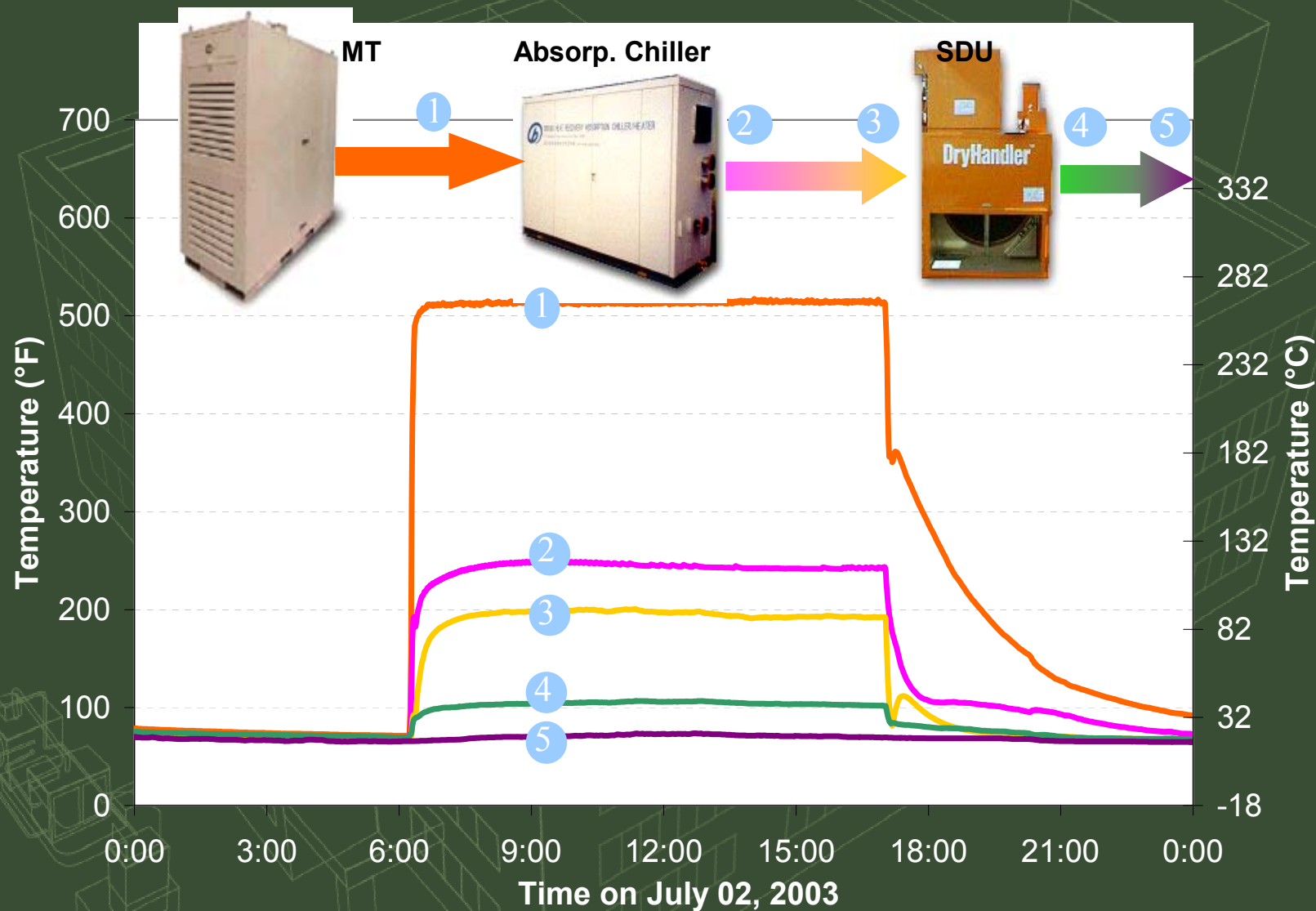
COP of Roof Top Unit

14



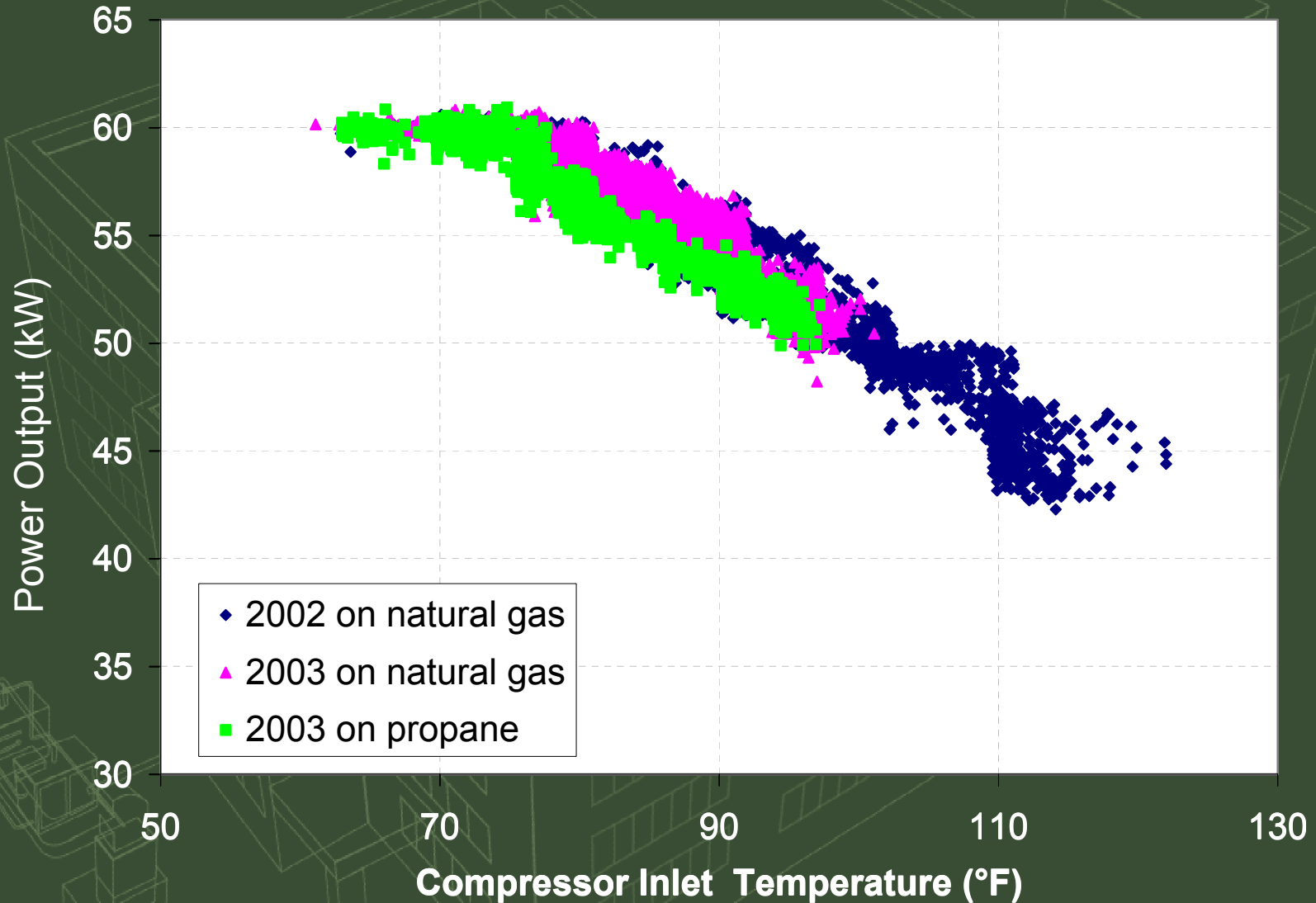
Temperature Profile of CHP System 2

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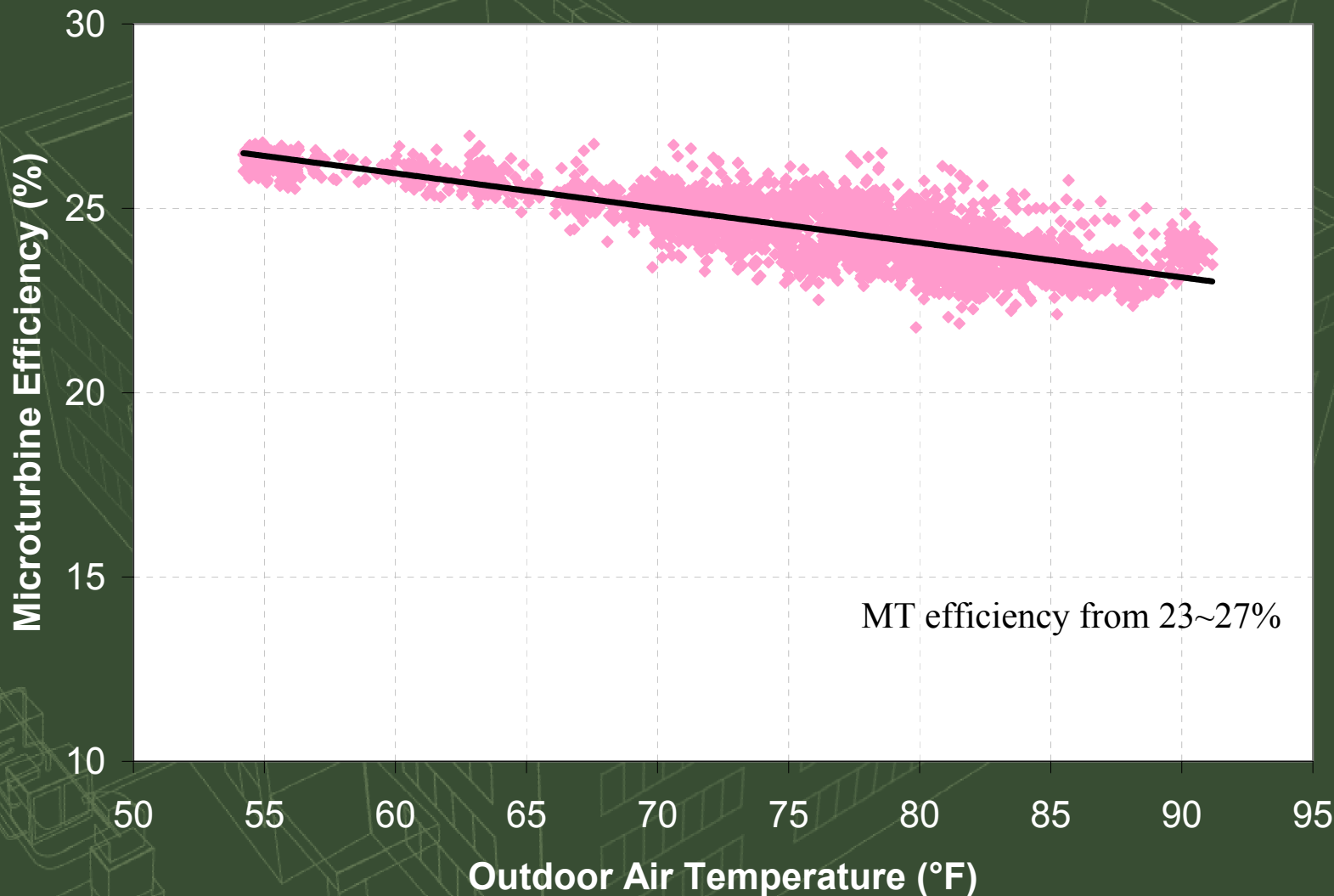
Microturbine Performance Comparison

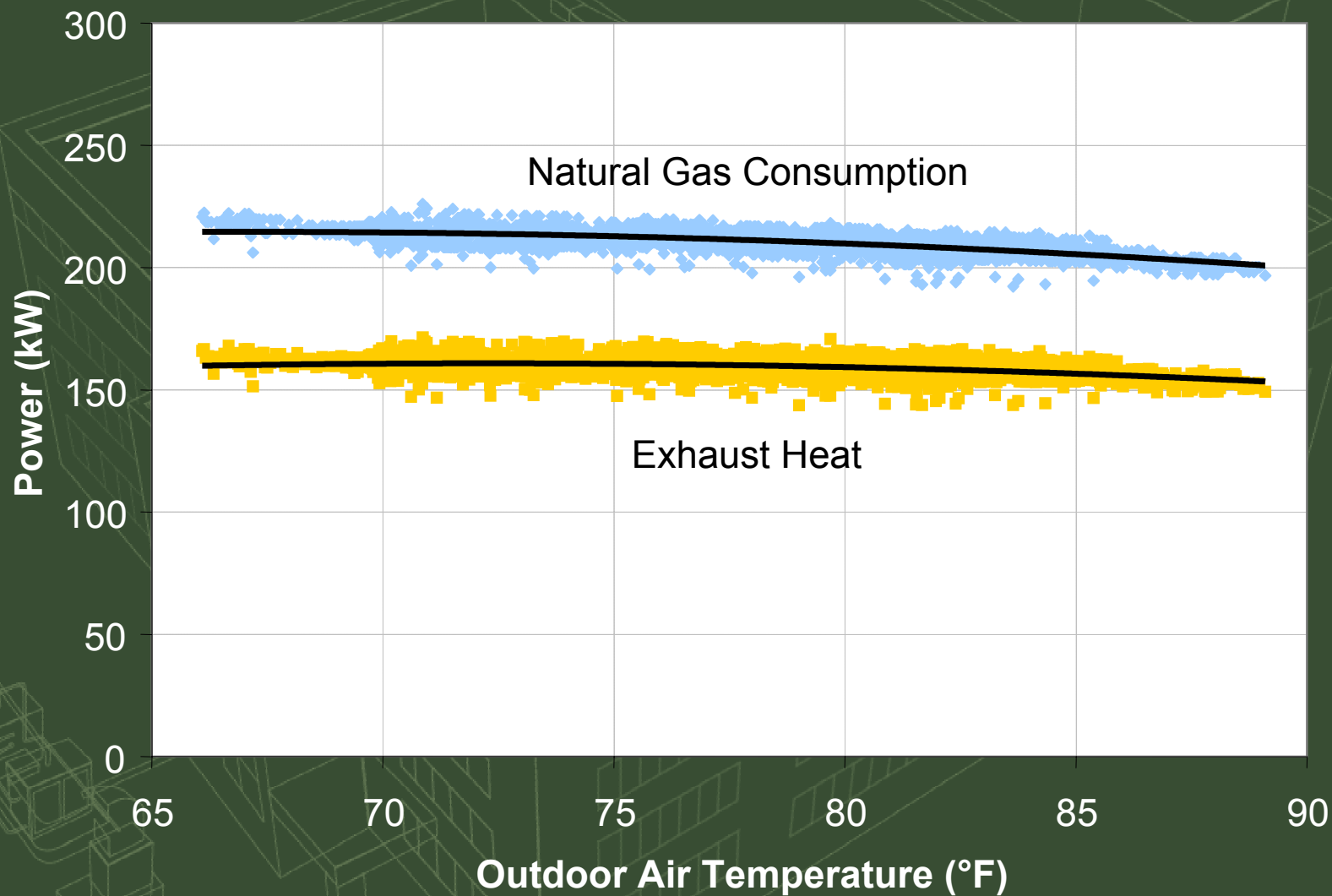
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Microturbine Performance Efficiency

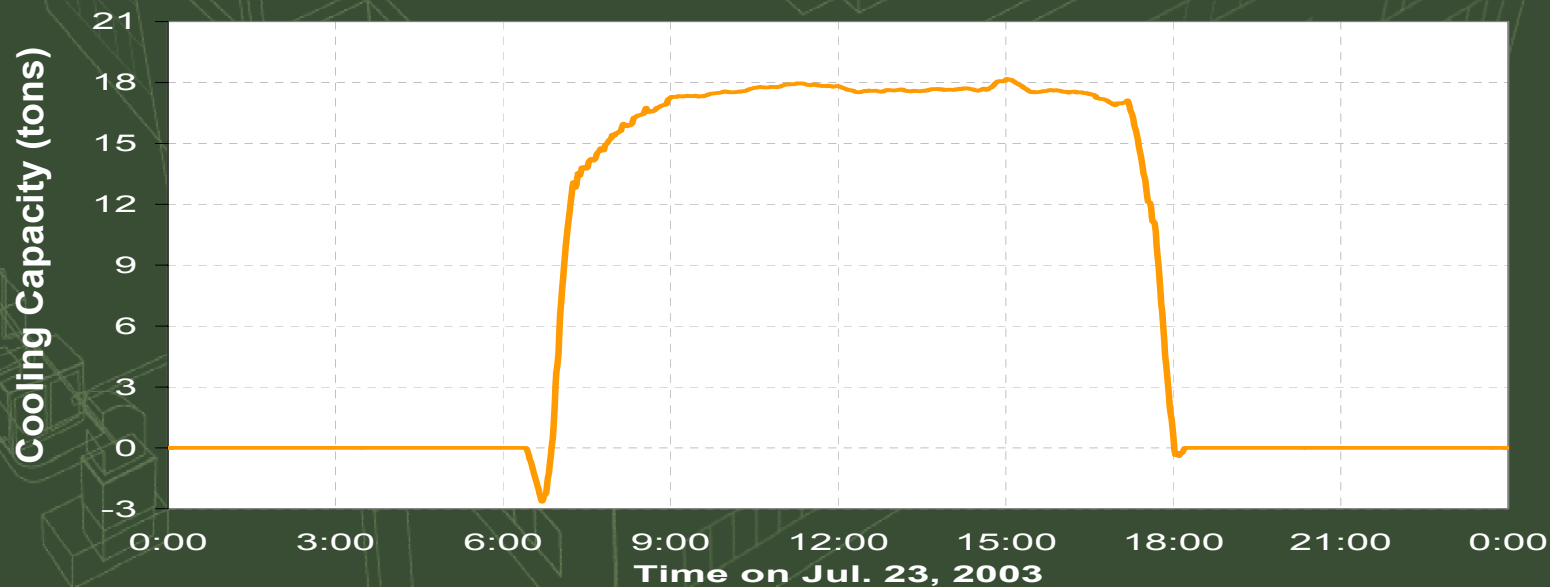
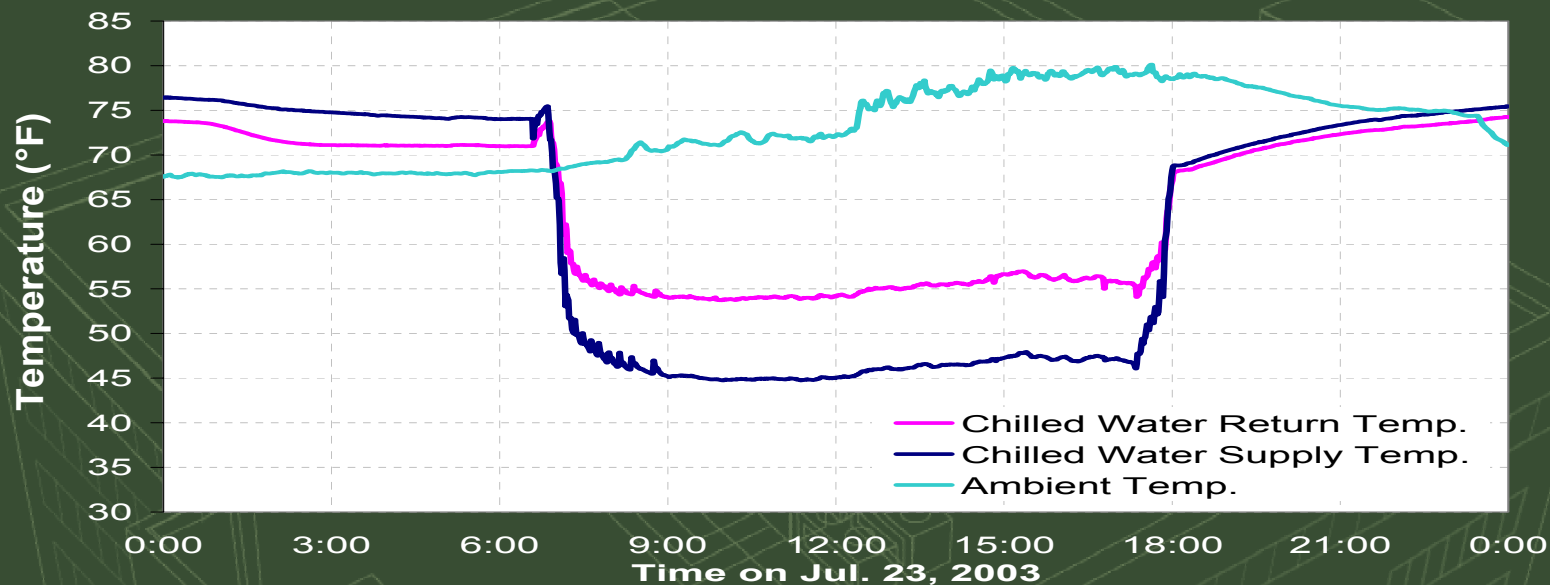
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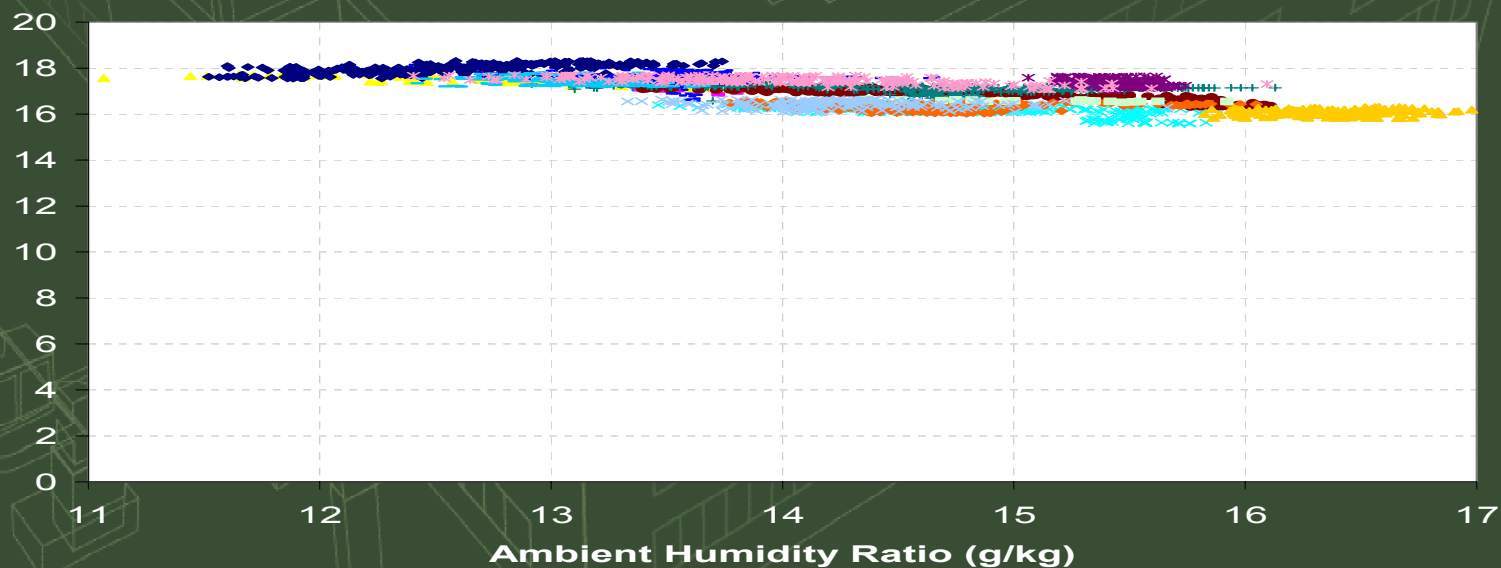
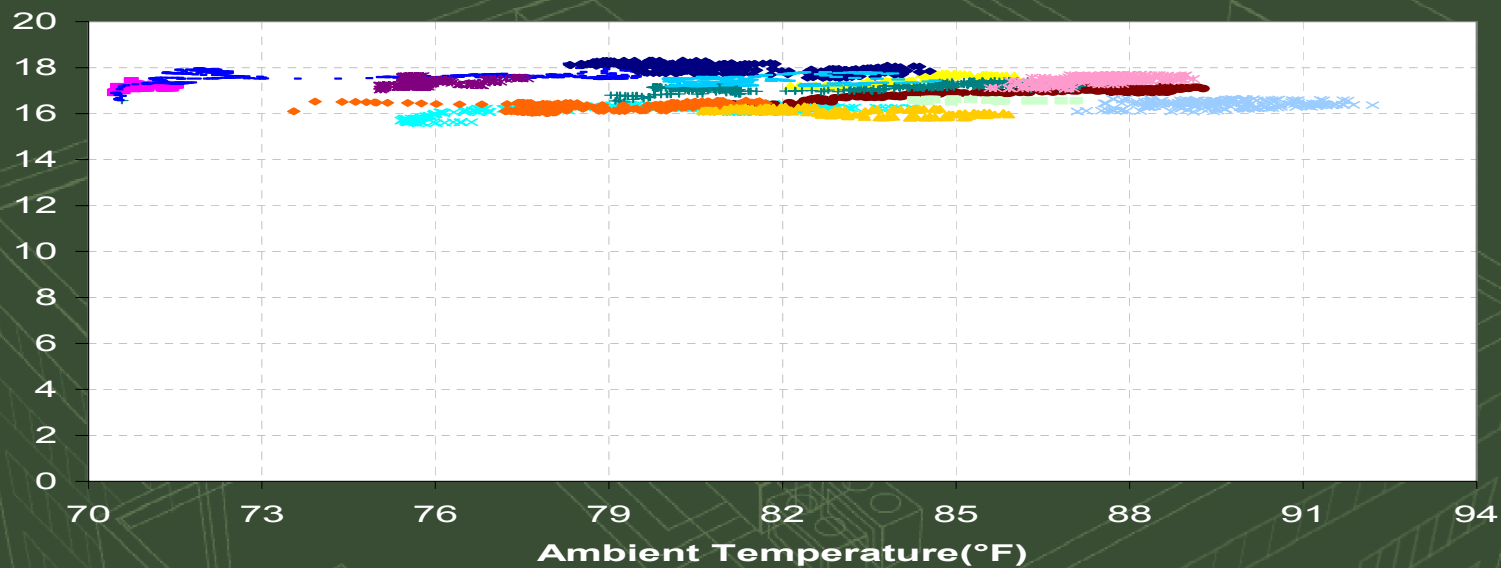
Chilled Water Temp. & Cooling Capacity

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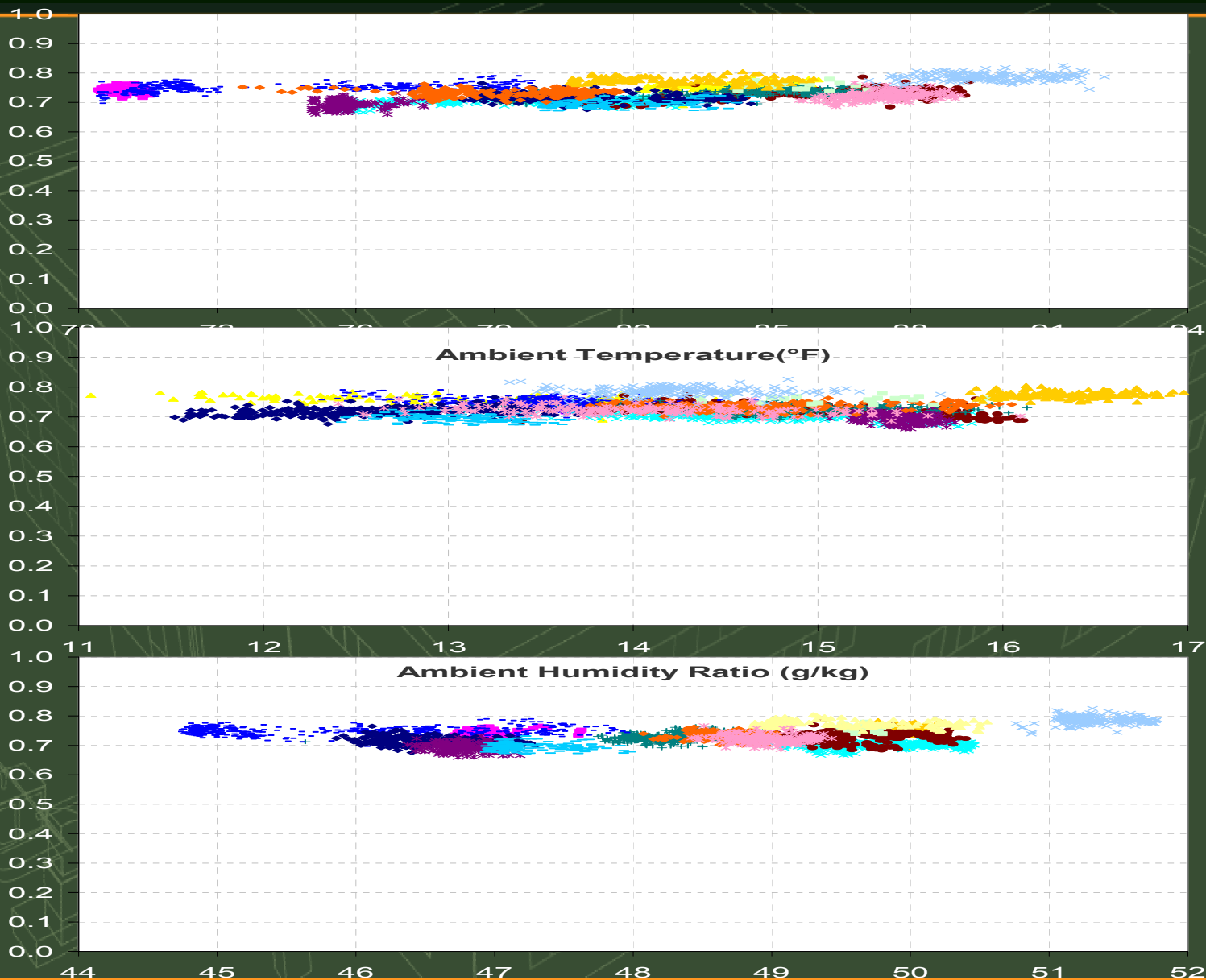
Absorption Chiller Cooling Capacity (tons)

20



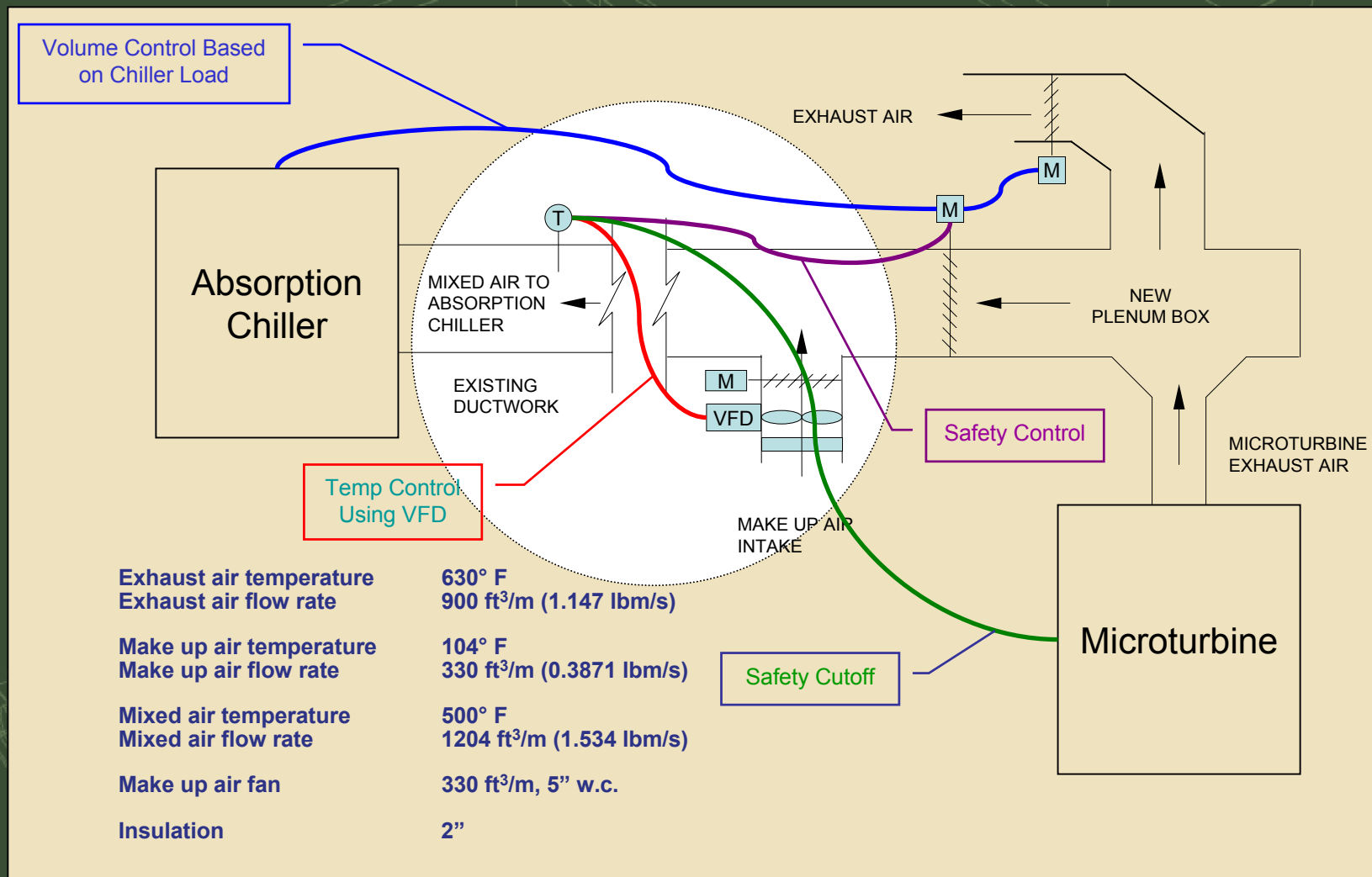
Absorption Chiller COP

21



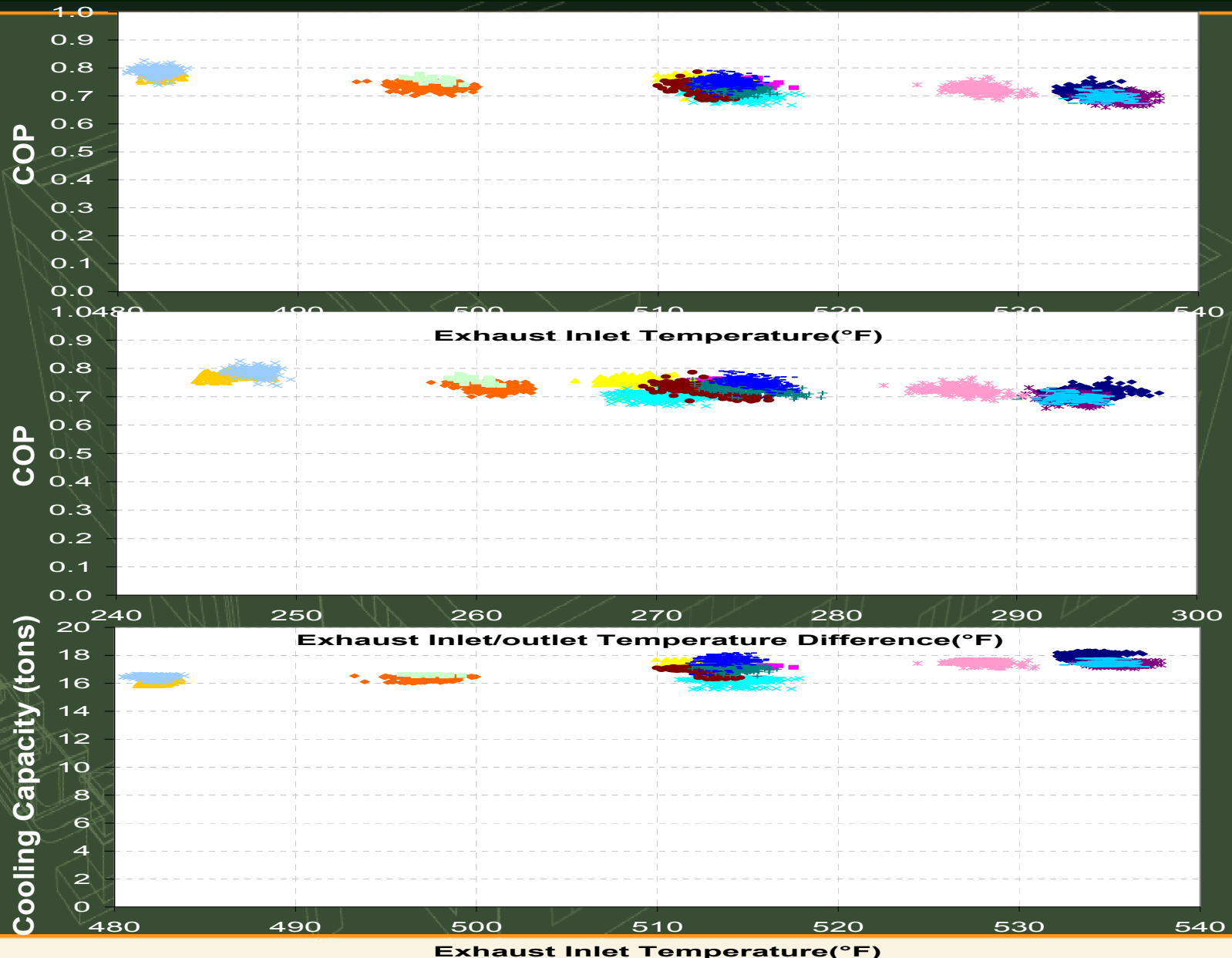
Exhaust Temperature Control

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Chiller Performance vs. Exhaust Inlet Temp.

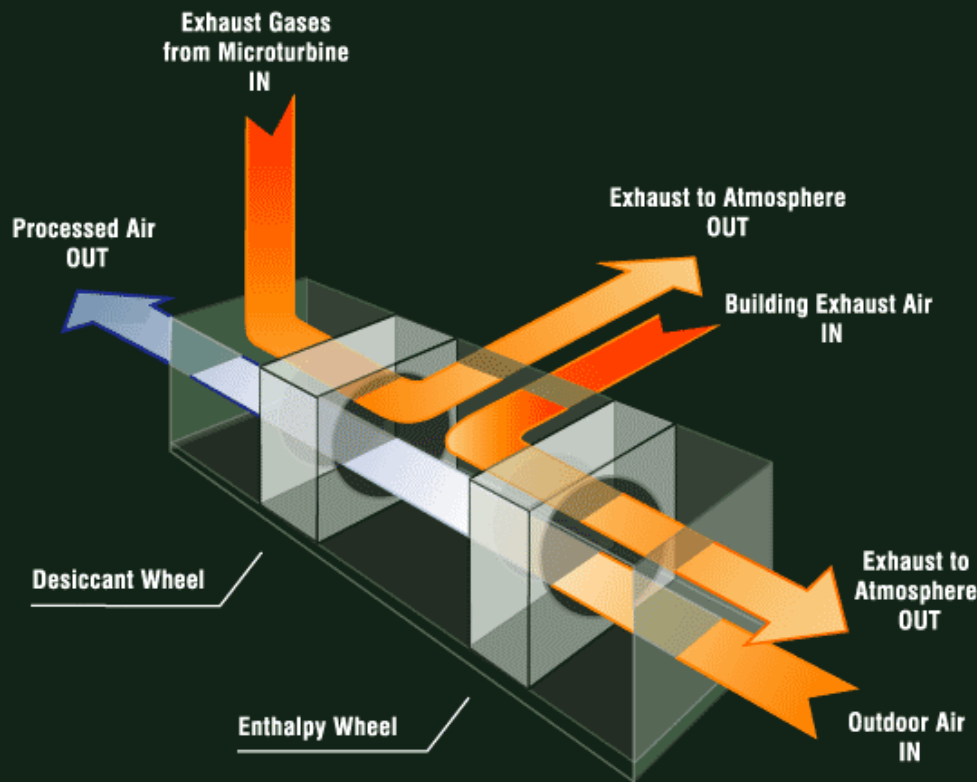
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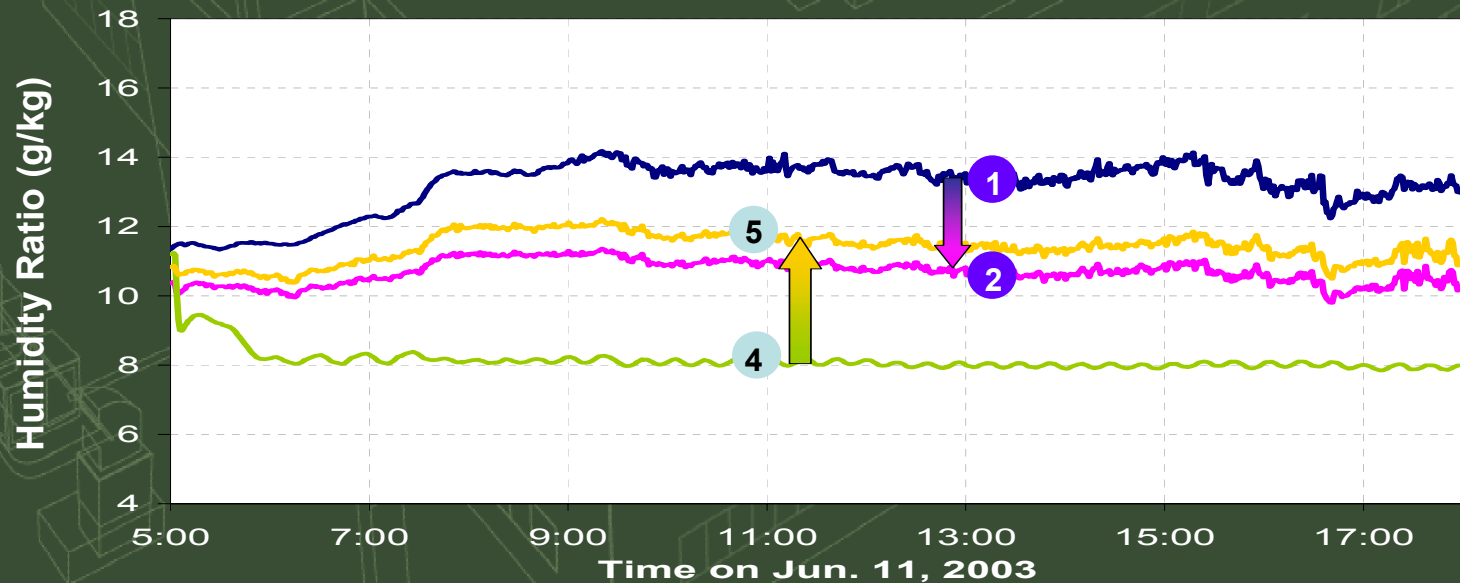
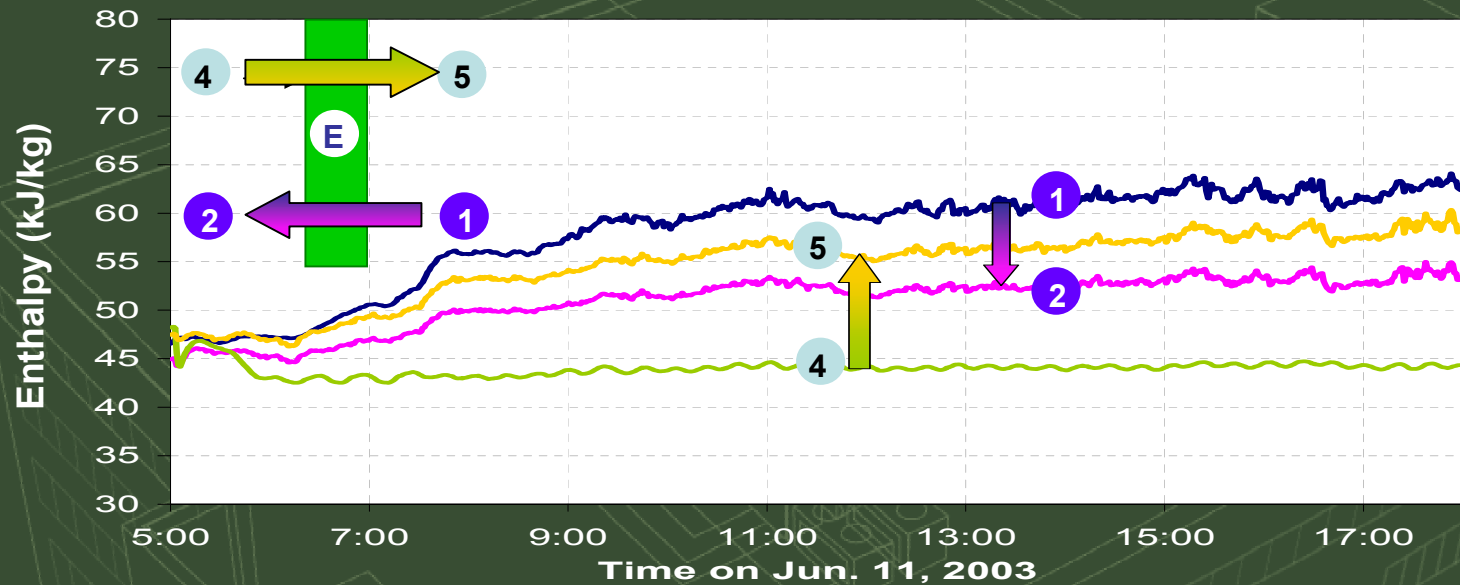


- COP is fairly constant regardless of the combined effect of weather, exhaust inlet temp. and chilled water supply temp.
 - $\text{COP}_{\text{thermal}} = 0.70$
- Mean chilled water supply temperature = 48°F
- Mean chilled water return temperature = 57°F
- Mean chilled water temp. difference = 9°F
- Mean chilled water flow rate = 47 gpm
- Mean Cooling Capacity ~ 18 tons.

Solid Desiccant CHP System ②

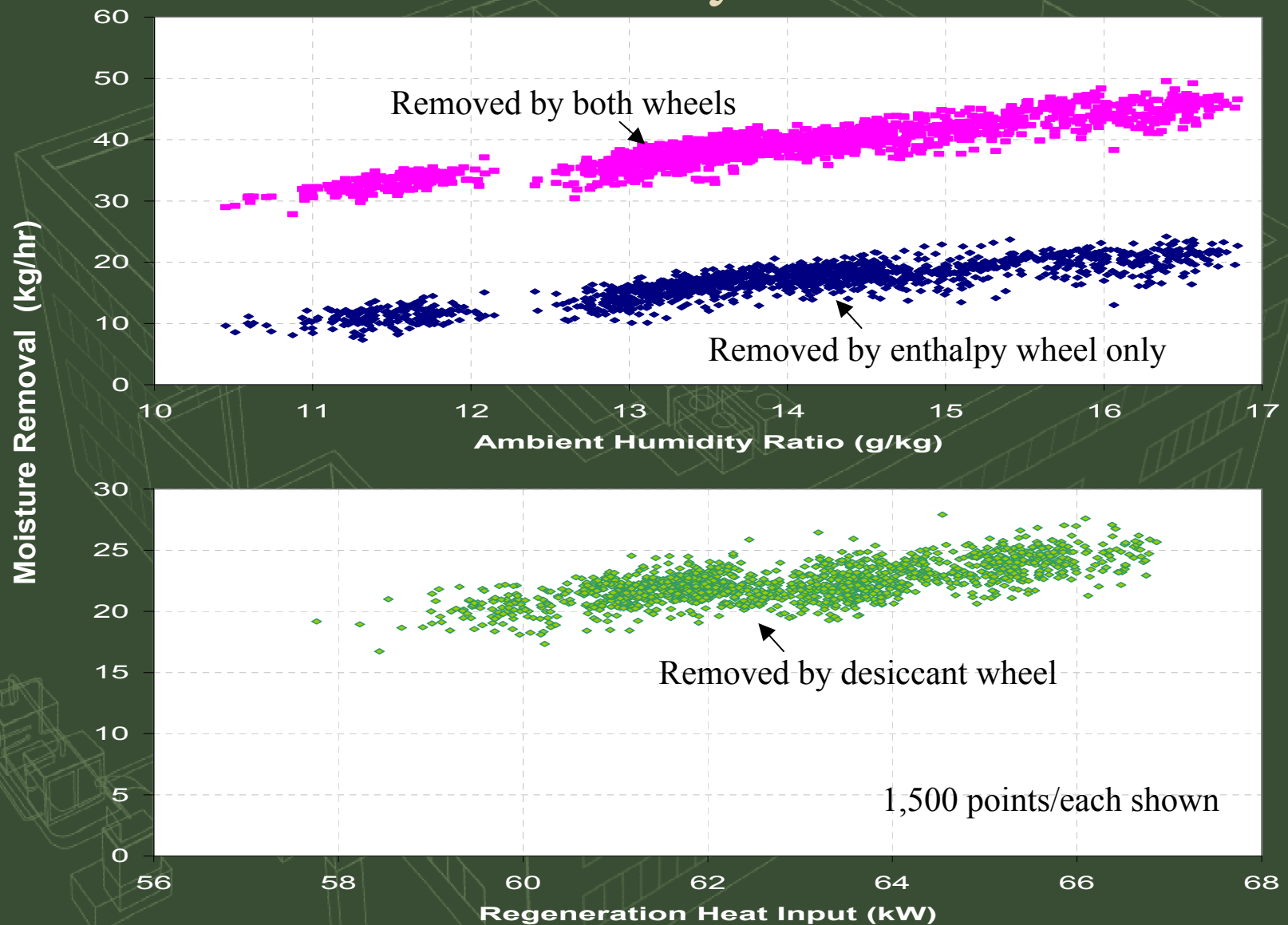
- Testing for performance difference between components – enthalpy wheel change out





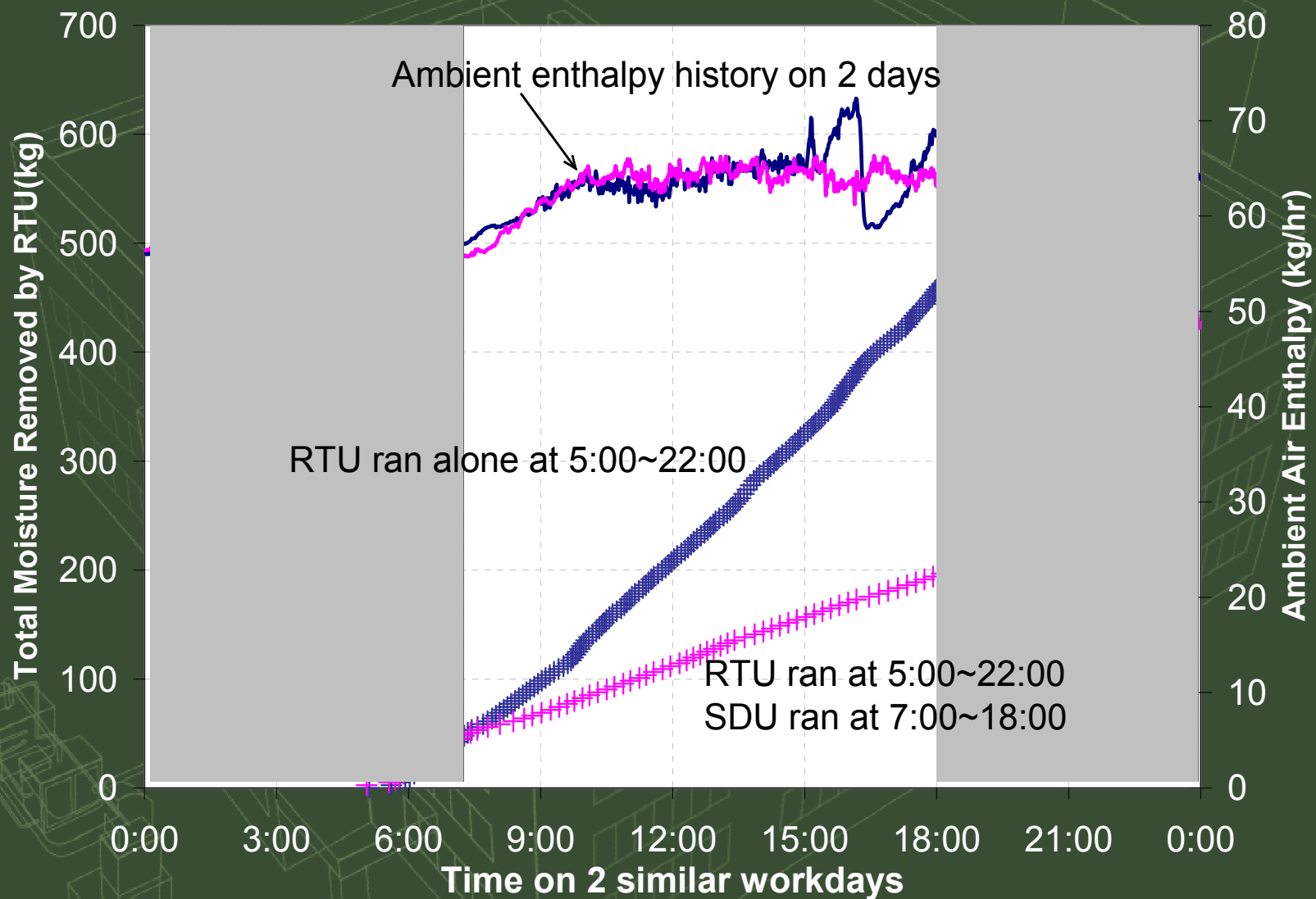
Solid Desiccant Unit Moisture Removal Ability

27



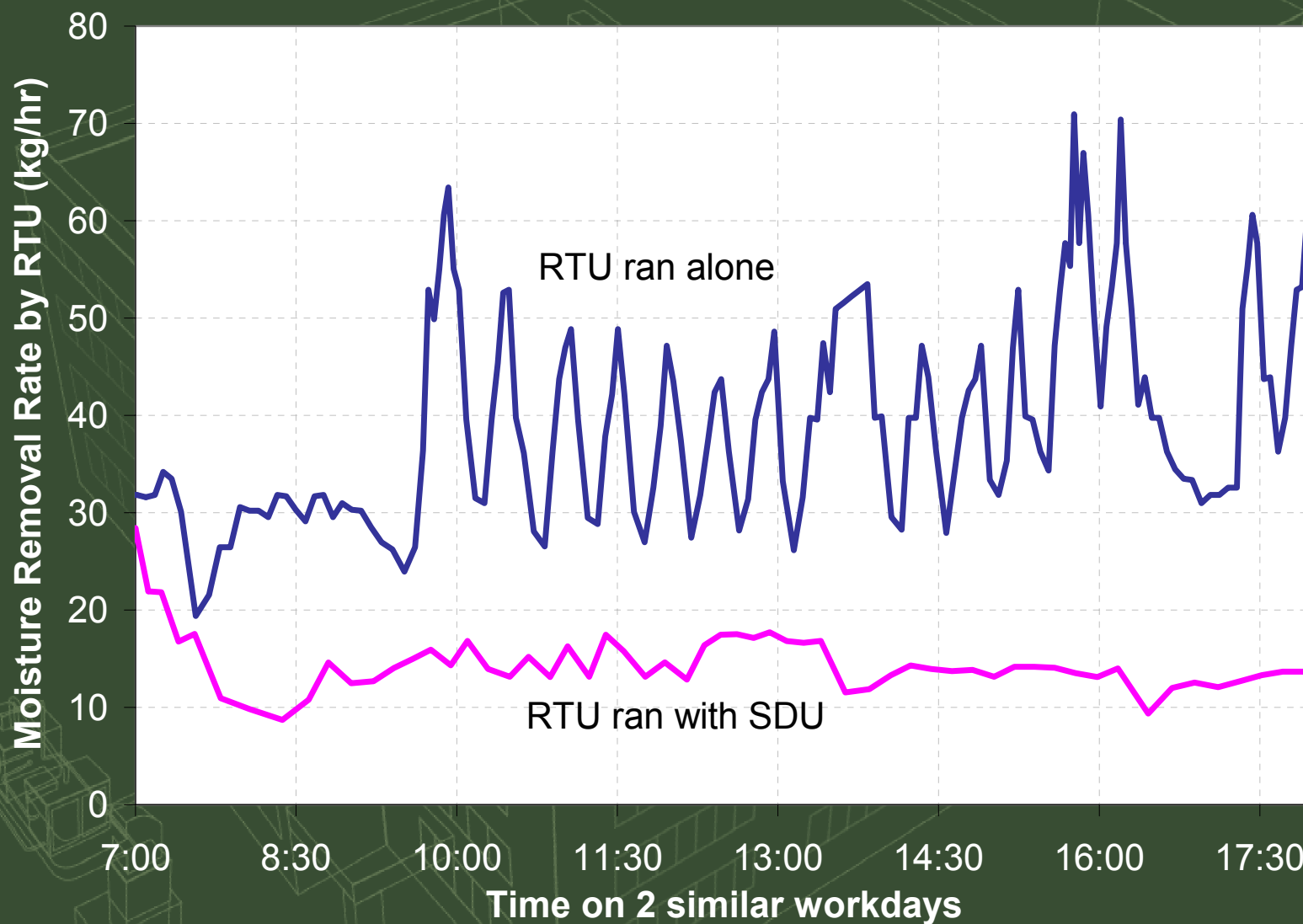
Comparison of Moisture Removal by RTU

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Moisture Removal Rate by RTU

29

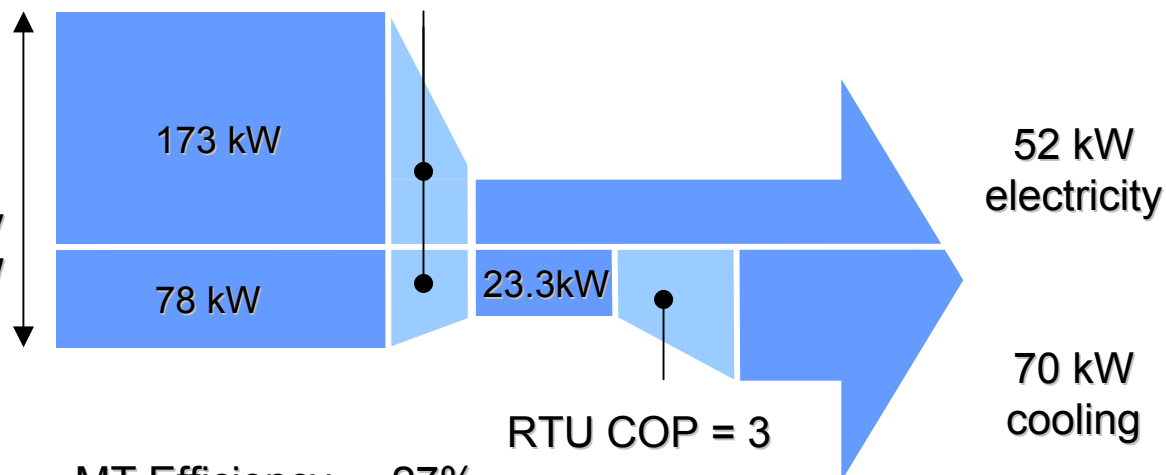


- Nominal output of CHP System 2:
 - 60kW power, 18 tons cooling, 3000CFM dry air
- Efficiency
 - MT 23%~27%
- Effectiveness of Enthalpy wheel
 - 0.75 for Original Wheel, 0.84 for New Wheel
- Desiccant wheel
 - COP latent 0.55
- COP
 - Absorption chiller: COP thermal 0.70
 - RTU COP 2.2~4.8

Grid Electric System

Primary Energy Consumption = 251 kW

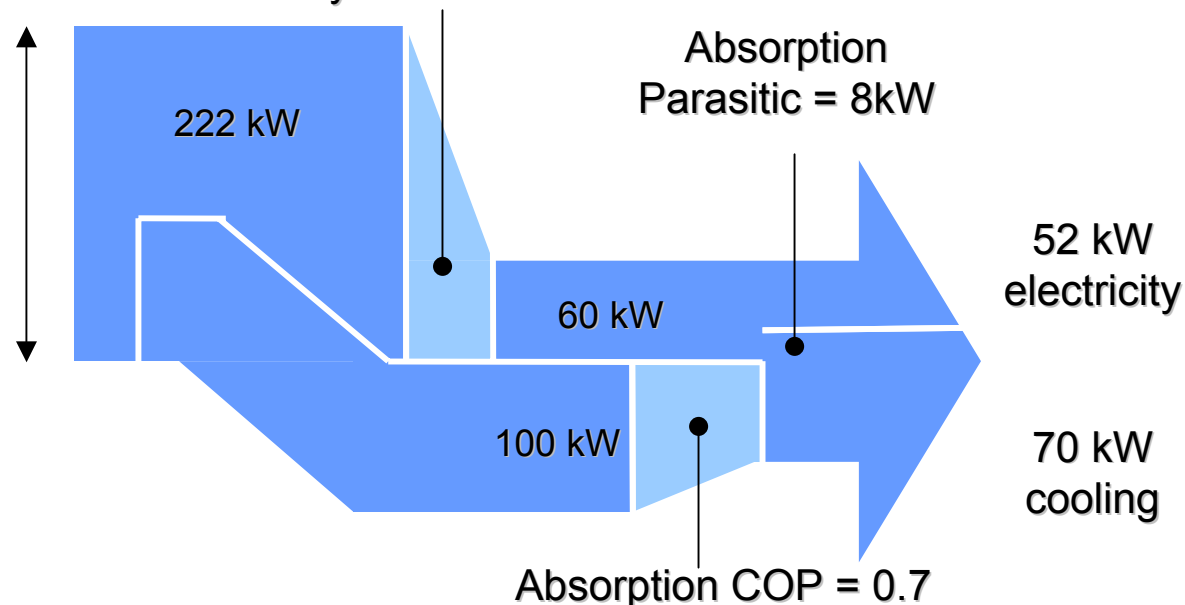
Grid Efficiency = 30%



Chesapeake System

Primary Energy Consumption = 222 kW

MT Efficiency = 27%



Overall Energy Savings:

RTU COP of 3.8 = 5%

RTU COP of 3 = 13%

RTU COP of 2.5 = 17%

- CHP integrates dissimilar equipment
- Components are generally designed to do one job well
 - MT produce power
 - Desiccants dry air
 - Absorption chillers produce chilled water
- Many additional benefits are obtainable when design is aimed at system level from start
- We want an INTEGRATED SYSTEM that is clean, reliable, efficient and cost effective

- Exhaust Gas Isolation
 - Protect downstream components from upstream heat production
 - Microturbine Exhaust Dampers – maximum leakage for dampers is 1%

**Crystallized
Absorption
Chiller**



**Exhaust
Damper
Leaked**

- Currently adopted by Capstone, UTC Power, Bowman, Broad, GTI, SoCal Gas

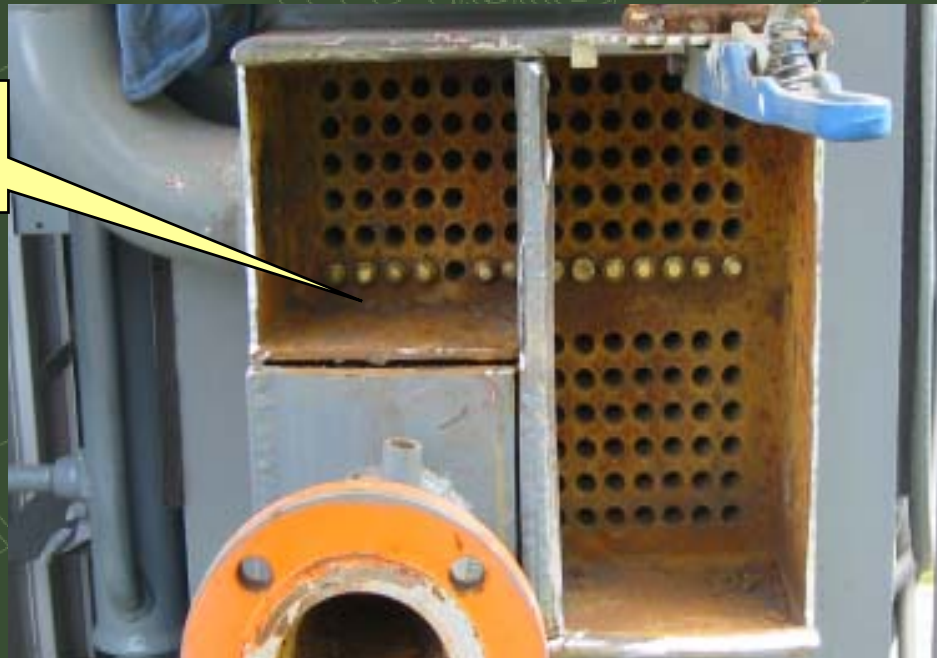
- Thermal Management
 - Modular damper manifold allowed for heat buildup in microturbine cabinet causing performance problems
 - Identification of this integration problem led to a redesigned manifold



- Ambient Protection

- CHP systems are generally located outdoors, but absorption chillers traditionally are not. The cold winter led to condenser tube damage which is currently being assessed to develop necessary design changes.

**Tubes Ruptured
or damaged due
to water freezing**



- TAT Integration



Conventional Field Integration



Newly Developed Packaged TAT

- Parameter compatibility
 - Exhaust temperatures vs. waste heat temp. requirements
- Standardization
 - Metric bolts, English nuts, specialist wiring harness tools, star-shaped sockets
 - Transformers, Fuses (5,12,24,120,230,277,480V AC/DC)
- Reduce duplication
 - Sensors
 - Enclosures
 - User Interfaces
 - Controllers and software drivers
- Maintenance Contracts, Manuals

- 200 Visitors to date in 2003 in 30 separate groups from conferences, governments, manufacturers, end users and students
 - David Garman, Assistant Secretary of Energy
 - International Congress of Refrigeration
 - Micro CHP Conference
 - Gas Cooling Workshop
- Trigen sponsors 8 undergraduate students and 2 graduate students to study CHP
- 1 Ph.D and 3 Master students graduated.



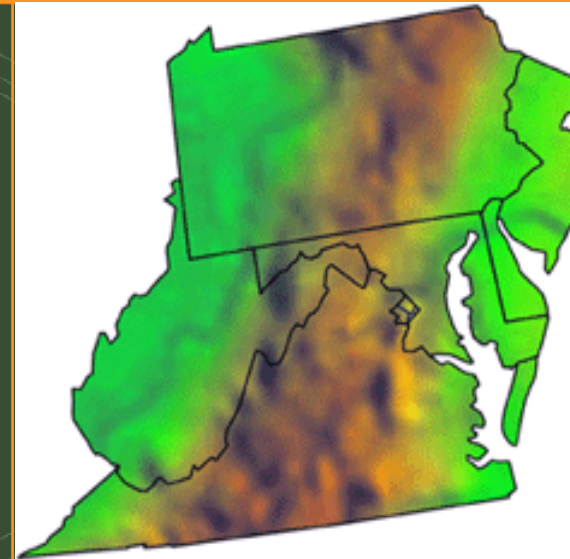
*David Garman
visited UMD on May
9, 2003*

- CHP courses
 - One annual undergraduate course
 - One biannual graduate course
- Presenting results and experience at
 - Semiannual consortium meetings
 - Attended/Contributed to 10 Conferences in 2003
 - Presented Workshop on CHP at International Congress of Refrigeration
- 4 papers published in 2003.



*Semiannual
consortium meetings*

- Based at University of Maryland
- Goal: Increase CHP Capacity in the Region
- How do we get there?
 - State baseline assessments
 - Partnership building (federal, local, private organizations and manufacturers, ESCOs and utilities)
 - “How to CHP” guidebook production
 - Promote sensible regulations
 - Outreach through website, brochures and conferences
 - Organize workshops



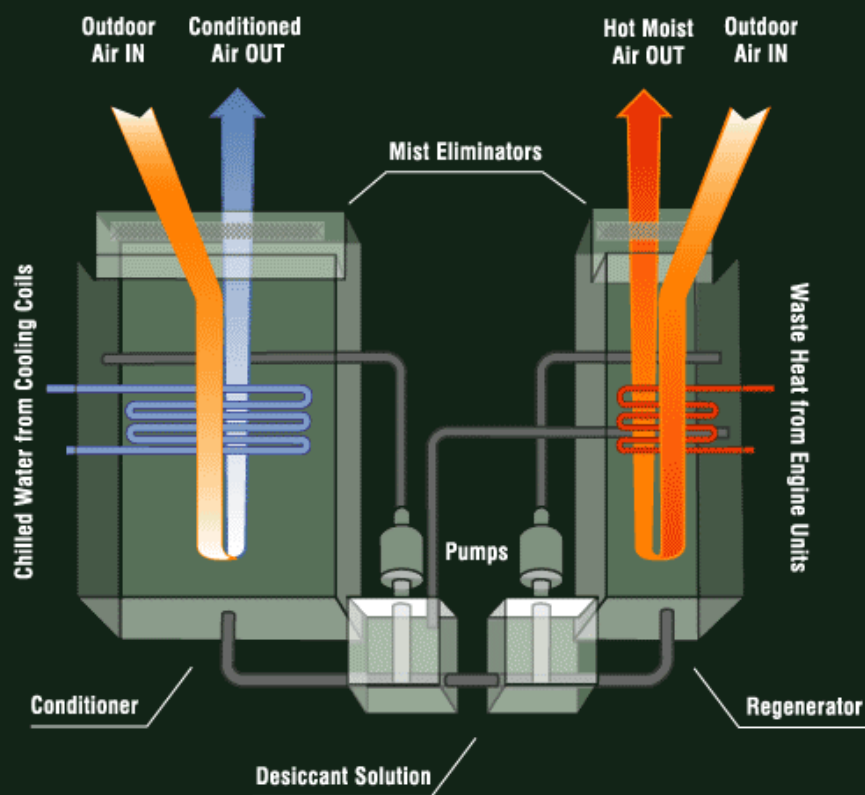
The 7 states covered are NJ, PA, DE, MD, VA, WV and DC



- Trigen CHP optimization
 - 27MW CHP plant on campus
 - Modeled and optimized in Thermoflex software
- MD IOF Energy Evaluations
 - Grant to visit industrial facilities in MD
 - Development of related research projects



Liquid Desiccant CHP System ①

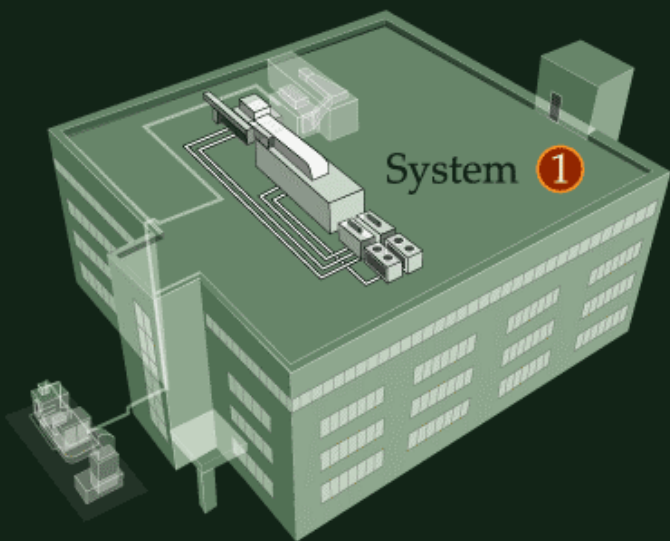


- New engine generator, integrated with liquid desiccant unit
- New microturbine & compact absorption chiller
- Transient modeling
- Heat pipe after desiccant wheel
- Ph.D dissertations
 - air-cooled absorption chiller
 - engine liquid desiccant system integration and performance optimization

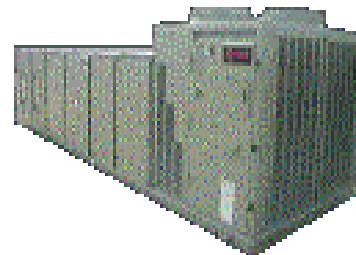
- New Genset to replace Engine Driven Air Conditioning Units
- DTE Energy and iPower pre-production model of 75kW reciprocating natural gas engine
- Packaged Heat Recovery – both engine jacket water and exhaust gases
- Provides hot water to regenerate liquid desiccant for CHP System 1



*The new DTE 75kW
packaged CHP
generator*



**EXISTING ROOF
TOP UNIT 1**



**3000 cfm of
dry air**

**75 kW of Electrical
Power**

**KATHABAR
LIQUID
DESICCANT**



**DTE 75kW
Reciprocating Engine
Genset with integrated
Heat Recovery**



**260 kW
Natural Gas**

**90 kW waste
heat @ 70°C
(water/glycol)**

- System 2 delivers stable and reliable data
- System 1 in the process of being changed
- Large amount of calibrated, quality field data collected and analyzed
- Fuel flexibility is demonstrated. Micro turbine runs on propane without problem.



Test facility at UMD